Charles's Law

T1 x V2 = T2 V1

or

T1 x P2 = T2 P1

PNEUMATIC APPLICATION and REFERENCE HAND BOOK

Cylinder Force

F = A x PSI

INNOVATION

Advanced Technology

Boyle's Law

P1 V1 x P2 V2

MEAD

Pneumatic Automation Components

Mead Fluid Dynamics is happy to bring you the Mead Pneumatic Application and Reference Handbook. It is loaded with helpful information regarding you fluid power application needs. Since the information in the booklet is an accumulation from several sources, please use this booklet as a reference only. Because we do not know the details of your application, Mead Fluid Dynamics assumes no responsibility or liability for your specific application. We hope you find the reference helpful.

MEAD FLUID DYNAMICS

History

In 1939 MEAD FLUID DYNAMICS was established in Chicago, Illinois. For over 65 years fluid power and industrial automation has evolved into our focus and core business. Mead has long been a leader in the development of award winning pneumatic components. The pneumatic side of fluid power is growing and is the backbone of American manufacturing. Mead Fluid Dynamics is positioned to be a part of that dramatic growth. Mead Fluid Dynamic firsts include:

First air clamp - 1943
First air switch - 1958
First all-pneumatic built-in, end-of-stroke sensor - 1960
First cylinders with hard-coated aluminum tubing - 1965
First snap-acting 4-way air valve - 1971
First totally integrated all-pneumatic, anti-tie down control - 1975
First non-lubricated cylinder - 1977
First valve to utilize multi-patented Isonic half-shell technology - 1992
First integrated manifold with snap-in valve - 1997
First push to connect/disconnect, multi-valve manifold - 2002

Mead USA

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Website: www.mead-usa.com e-mail: sales@mead-usa.com

Mead Customer Service and Technical Service are available to help with the selection of product and system design related questions.

Mead Fluid Dynamics has a network of over 70 authorized distributors in the USA and over 100 worldwide. Many distributors stock products for quick delivery. Most products are delivered in a few days. Check our web site for our latest products, dimensional drawings and other application tips.

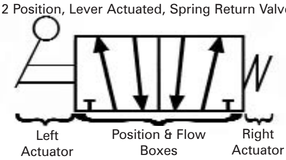
Today, proper air preparation is mandatory. A high level of air filtration (at least 3 micron) is always a good investment. The use of coalescesing filtration is highly recommended.

Table of Contents

Understanding Circuit Symbols

Directional air control valves are the building blocks of pneumatic control. Symbols representing these valves provide a wealth of information about the valve it represents. Symbols show the methods of actuation, the number of positions, the flow paths and the number of ports. Here is a brief breakdown of how to read a symbol:

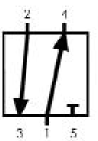
2 Position, Lever Actuated, Spring Return Valve



Every symbol has three parts (see figure to right). The Left and Right Actuators are the pieces which cause the valve to shift from one position to another. The Position and Flow Boxes indicate how the valve functions. Every valve has at least two positions and each position has one or more flow paths.

When the Lever is not activated, the Spring Actuator (right side) is in control of the valve; the box next to the actuator is the current flow path. When the Lever is actuated, the box next to the Lever is in control of the valve. Each position occurs when the attached actuator is in control of the valve (Box next to the actuator). A valve can only be in one “Position” at a given time.

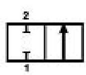
The number of boxes that makes up a valve symbol indicates the number positions the valve has. Flow is indicated by the arrows in each box. These arrows represent the flow paths the valve has when it is that position (depending upon which actuator has control over the valve at that time).

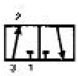


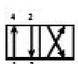
The number of ports is determined by the number of end points in a given box (only count in one box per symbol as the other boxes are the just showing different states of the same valve). In the example, there are a total of 5 ports. NOTE: Sometimes a port (such as exhaust) goes directly to atmosphere and there is no port to attach to. To spot this, the actual ports line will extend beyond the box, while the ports you cannot attach to will not. A Port is blocked with this symbol: ┬

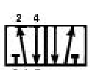
Following is a list of symbols and what they mean:

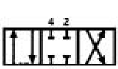
Valve Symbols, Flow Paths and Ports


2-Position, 2-Way, 2-Ported

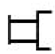

2-Position, 3-Way, 3-Ported

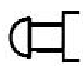

2-Position, 4-Way, 4-Ported

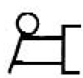

2-Position, 4-Way, 5-Ported

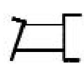

3-Position, 4-Way, 4-Ported
Closed Center

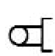
Actuator Symbols


Manual

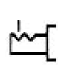

Push Button

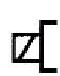

Lever


Foot Operated


Mechanical

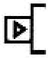

Spring

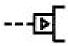

Detent

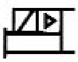

Solenoid

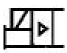
Symbols Continue on Next Page

Actuator Symbols


Internal Pilot


External Pilot


Piloted Solenoid with Manual Override

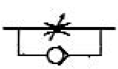

Piloted Solenoid and Manual Override

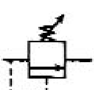

Lever Operated, Spring Return


Solenoid Operated, Spring Return

Simple Pneumatic Valves


Check Valve

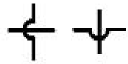

Flow Control, 1 Direction

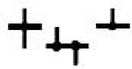

Relief Valve

Lines

Main Line

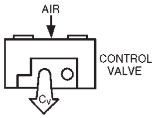
Pilot Line


Lines Crossing


Lines Joined


Lines Joined

C_V Defined



Q: What does "C_V" mean?
A: Literally C_V means coefficient of velocity. C_V is generally used to compare flows of valves. The higher the C_V, the greater the flow.

It is sometimes helpful to convert C_V into SCFM(Standard Cubic Feet per Minute) and conversely, SCFM into C_V. Although C_V represents flow capacity at all pressures, SCFM represents flow at a specific air pressure. Therefore, the following chart relates C_V to SCFM at a group of pressures.
To obtain SCFM output at a particular pressure, divide the valve C_V by the appropriate factor shown below.

Cv to SCFM Conversion Factor Table							
PSI of Air Pressure	40	50	60	70	80	90	100
Factor	.0370	.0312	.0270	.0238	.0212	.0192	.0177

Example: What is the output in SCFM of a value with a C_V of 0.48 when operated at 100 PSI?

$$\frac{0.48(CV)}{.0177(Factor)} = \mathbf{27\ SCFM}$$

To convert SCFM into C_V, simply reverse the process and multiply the SCFM times the factor.

Pneumatic Valve Sizing

Two methods are shown below to aid in the selction of a pneumatic valve. To account for various losses in all pneumatic systems, remember to over-szie by at least 25%.

Method 1: Calculation

This formula and chart will give the C_v (Valve flow) required for operating a given air cylinder at a specific time period.

$$C_v = \frac{Area \times Stroke \times A \times C_f}{Time \times 29}$$

Area = π x Radius ^{2} or see table B below.
Stroke = Cylinder Travel (in.)
A = Pressure Drop Constant (see table A)
C_f = Compression Factor (see table A)
Time = In Seconds

Table A

Inlet Pressure (PSI)	Cf Compression Factor	"A" Constants for Various Pressure Drops		
		2 PSI ΔP	5 PSI ΔP	10 PSI ΔP
10	1.6		0.102	
20	2.3	0.129	0.083	0.066
30	3.0	0.113	0.072	0.055
40	3.7	0.097	0.064	0.048
50	4.4	0.091	0.059	0.043
60	5.1	0.084	0.054	0.040
70	5.7	0.079	0.050	0.037
80	6.4	0.075	0.048	0.035
90	7.1	0.071	0.045	0.033
100	7.8	0.068	0.043	0.031
110	8.5	0.065	0.041	0.030
120	9.2	0.062	0.039	0.029

Table B

Bore Size	Cylinder Area (Sq. In.)
1/4"	0.049
1/2"	0.196
3/4"	0.44
1-1/8"	0.99
1-1/2"	1.77
2"	3.14
2-1/4"	3.97
2-1/2"	4.91
3"	7.07
3-1/4"	8.30
4"	12.57
5"	19.64
6"	28.27
8"	50.27
10"	78.54
12"	113.10

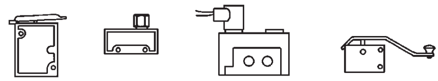
NOTE: Use "A" Constant at 5 PSI ΔP for most applications. For critical applications use "A" at 2 PSI ΔP. A 10 PSI ΔP will save money and mounting space.

Method 2: Chart

Index C_v against Bore Size vs. Inches of stroke per second.
Assuming 80 PSI and ΔP = 80%.

C_v	Cylinder Bore Size									
	0.75	1.13	1.50	2.00	2.50	3.25	4.00	5.00	6.00	8.00
0.1	26.8	11.9	6.7	3.8	2.4	1.4	0.94	0.6	0.42	0.24
0.2	53.7	23.9	13.4	7.5	4.8	2.9	1.9	1.2	0.84	0.47
0.5	134	59.6	33.6	18.9	12.1	7.1	4.7	3	2.1	1.2
1.0	268	119	67.1	37.7	24.2	14.3	9.4	6	4.2	2.4
2.0	537	239	134	75.5	48.3	28.6	18.9	12.1	8.4	4.7
4.0		477	268	151	96.6	57.2	37.7	24.2	16.8	9.4
8.0			536	302	193	114	75.5	48.3	33.6	18.9
16.0				604	387	229	151	96.6	67.1	37.7
32.0					773	457	302	193	134	75.5

Valve Selection



Q: How do I select the right valve to control a cylinder?
A: There are many factors that contribute to the performance of a cylinder. Some of these factors are: quantity and type of fittings leading to the cylinder, tube length and capacity, cylinder operating load, and air pressure.

Rather than attempting to place a value on these, and other contributing factors, it is more practical to provide valve users with a general guide to valve sizing.

The sizing table below relates various Mead air valves to cylinder bore sizes between 3/4″ and 6″ . The cylinder operating speed resulting from the use of each valve at 80 PSI is rated in general terms as:

"F" for High Speed Operation **"M"** for Average Speed Operation **"S"** for Slow Speed Operation

Valve Type	Cv	Cyl. Type*	Cylinder Bore Sizes (in inches) 3/4	1	1 1/8	1 1/2	2	2 1/4	2 1/2	3	3 1/4	4	6
Micro-Line	0.11	SA		F				S		S			
LTV	0.18	SA,DA	F	F	F	F	M	M	M	M	S	S	
Nova	1.00	SA,DA			F	F	F	F	F	F	F	M	M
Duramatic	0.18	SA,DA	F	F	F	F	M	M	M	M	S	S	
Duramatic	0.63	SA,DA			F	F	F	F	F	M	M	M	M
Capsula	0.75	SA,DA			F	F	F	F	F	F	M	M	M
Capsula	3.17	SA,DA									F	F	F
FT-1, FC-1	0.13	SA		F				F		M		S	
4B-1, 4W-1	0.48	SA,DA			F	F	F	F	M	M	M	S	
FC51, PC51	0.81	SA						F		F		M	M
FT-101, 201	1.15	SA						F		F		F	F
Isonic V1	0.01-0.05	SA	F	M									
Isonic V4	0.8	SA,DA			F	F	F	F	F	F	M	M	M
Isonic V3	0.03-0.11	SA	F	F	F	M	S						
Isonic V5	0.8	SA,DA			F	F	F	F	F	F	M	M	

* SA = Single-Acting Cylinder, DA = Double-Acting Cylinder

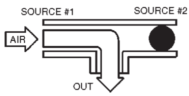
Where no rating is shown, the valve is considered unsuitable for use with that particular bore size. To determine the suitability of valves not listed in the table, compare the C_V of the unlisted valve with the one nearest it on the table and use that line for reference.

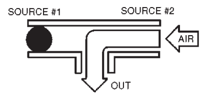
SCFM Defined

Q: What does SCFM mean?
A: SCFM means Standard Cubic Feet per Minute. “Standard” is air at sea level and at 70° F.

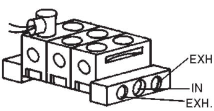
Shuttle Valves

Q: Is there a valve that will direct air coming from either of two sources to a single destination?
A: Use a shuttle valve.



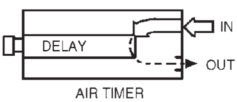


Stacking



Q: How may I reduce piping and simplify trouble-shooting when a group of valves is used in an application?
A: Order your valves stacked to take advantage of a common air inlet, common exhausts, and control centralization.

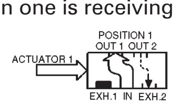
Time Delay

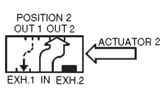


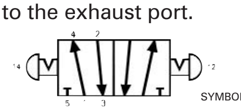
Q: Are there valves that allow me to delay a signal in my air circuit?
A: Yes, Mead air timers can be used to delay an air signal. Up to 2 minute normally open or normally closed models are available.

Two-Position - vs - Three-Position

Q: What is the difference between 2-position and 3-position valves?
A: In two-position four-way directional valves, the two output ports are always in an opposite mode. When one is receiving inlet air, the other is connected to the exhaust port.

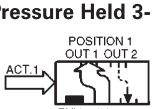


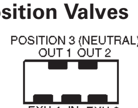


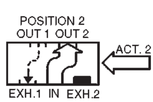


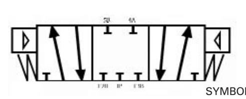
When actuated, 3-position 4-way directional valves function the same as above. However, a center or “neutral” position is provided that blocks all ports (pressure held), or connects both output ports to the exhausts (pressure released) when the valve is not being actuated.

Pressure Held 3-Position Valves



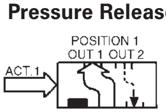


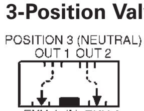


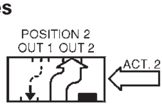


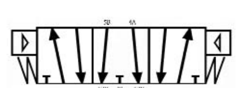
Pressure held models are ideal for “inching” operations where you want the cylinder rod to move to a desired position and then hold.

Pressure Released 3-Position Valves



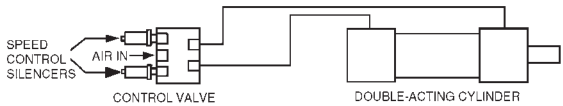




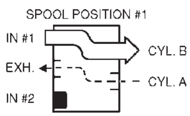


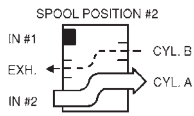
Five-Ported Valves

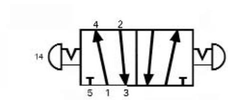
Q: What are the advantages of a five-ported four-way valve over a four ported four-way valve?
A: Five ported valves have separate exhaust ports for each cylinder port. If exhaust silencers with built-in speed controls are used, the speed of the cylinder motion may be individually controlled in each direction.



Also, five ported valves can function as dual pressure valves where air flows from the exhaust ports to the cylinder and both cylinder ports use the inlet as a common exhaust. Vacuum may also be used in five ported valves. Both the Mead Nova line and the Capsula line provide five ported flow patterns.

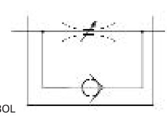






SYMBOL

Flow Control



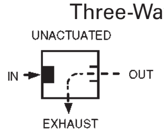
SYMBOL

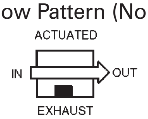
Q: Are there valves available that provide adjustable control of air flow?
A: Mead Dylatrol valves perform this function. Also see the "Cylinders; Speed Control" question for application information. Dura-matic directional valves have built-in flow controls. Exhaust silencers typically have built-in needle valves that also provide speed regulation. See the Mead catalog for more information.

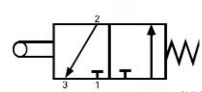
Flow Patterns, 3-Way & 4-Way

Q: What is the difference between a 3-way and a 4-way valve?
A: Three-way valves have one power output and four-way valves have two power outputs. Generally, three-way valves operate single-acting cylinders and four-way valves operate double-acting cylinders. For 3-Way and 4-Way valves, see the Mead Catalog MV and LTV valves. (Respectively)

Three-Way Flow Pattern (Normally Closed)







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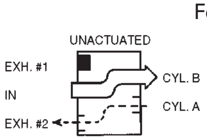
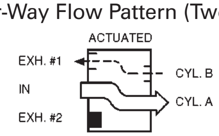
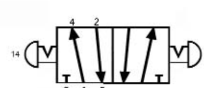
Four-Way Flow Pattern (Two Position)

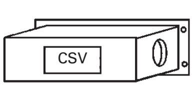






SYMBOL

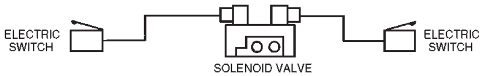
For Safer Hand Actuation



Q: How may I keep the hands of my employees out of hazardous locations?
A: Use two-hand, anti-tiedown devices.

Air -vs- Solenoid Actuation

Q: What are the advantages of air actuation over solenoid actuation?
A: Solenoid actuation requires the presence of electric switches, wires, and all of the shielding necessary to reduce spark hazard and personal risk.



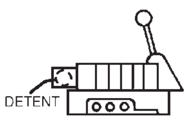
NOTE: The Solenoid Valve shown here is N2-DCD.

Air actuation requires only 3-way air pilot valves and tubing. There is no explosion, spark, or shock risk and the components are less expensive to buy.

NOTE: The Air Piloted Valve shown here is the N2-DP. The 3-Way Pilot Valves are from the MV series.

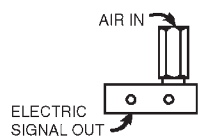
Detented Valves

Q: What is a "detented" valve and how is it used?
A: A detented valve is one that holds its position by some mechanical means such as a spring, ball or cam. Most valves hold their position by means of the natural friction of the rubber seals. Where natural friction is low, such as in packless valves, or where it is not enough for safety purposes, detented models are recommended. Also, detents are used to locate the middle position in three position valves. See the Capsula Valve Section in the Mead Catalog.



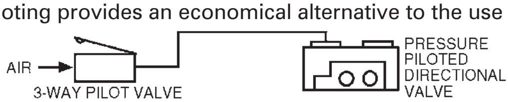
Air-To-Electric Signal Conversion

Q: Is it possible to convert an air signal into an electrical signal?
A: Mead air-to-electric switches, MPE-BZ or MPE-BZE (includes enclosure), will turn an air signal into an electrical signal, which can be wired either normally open or closed.



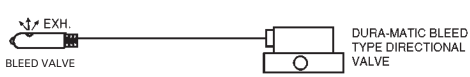
Pressure Piloted - vs - Bleed Piloted

Q: What is the difference between pressure piloted valves and bleed piloted valves?
A: Pressure piloting and bleed piloting refer to two different modes in which valves may be actuated. Pressure piloting positively actuates a directional valves by an external air signal that comes from a remote three-way valve, such as the Micro-Line valve series. Air pressure piloting provides an economical alternative to the use of electric switches and solenoids.



NOTE: Valves Shown here are from the Nova Series (Pressure Piloted Directional Valve) and the MV Series (3-Way Pilot Valve).

Bleed piloting uses internal air from the directional valve to feed the pilot valve. Air flows from the directional valve to the bleed valve. When the bleed valve is actuated, a pressure drop occurs in the directional valve pilot section. This causes a differential pressure and valve shift.



The main advantage of bleed piloting is that only one line enters the bleed valve. However, if the line is severed, a shift occurs. Pressure piloting is considered more positive and reliable.

Low Force To Actuate

Q: Are there valves available that require an unusually low force to actuate?
A: Low-stress valves need only 6-8 oz. of force to initiate a signal. These valves reduce stress on worker's hands. LTV four-way valves operate on a pressure differential basis that allows them to actuate on very little force.

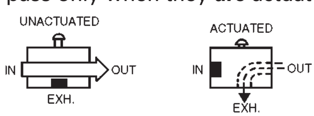
Manual Overrides

Q: What are manual overrides in air valves used for?
A: Manual overrides permit the user to actuate the directional valves without using the switches or pilot valves that would normally be used. In this way, a circuit may be tested without actually moving the machine elements.

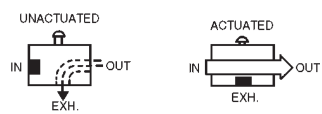
Both Capsula valves and Nova valves are available with manual overrides.

Normally Closed - vs - Normally Open

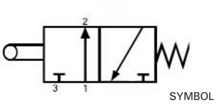
Q: What is the difference between a three-way normally closed valve and a three-way normally open valve?
A: Normally open valves allow air to pass when **not** actuated. Normally closed valves allow air to pass only when they **are** actuated.

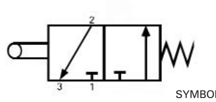


Normally Open Flow Pattern

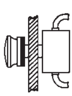


Normally Closed Flow Pattern



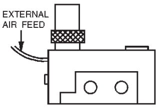


Panel Mounted



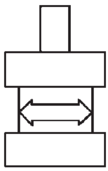
Q: Are there valves available that fit through "knockouts" in control panels?
A: MV 3-way valves and LTV 4-way valves have threaded mounting stems for panels.

External Air Supply For Solenoids



Q: Under what conditions should an external air supply be used to feed the solenoids on a directional valve?
A: When the air pressure passing through the power section of the valve is insufficient to shift the spool, when the medium passing through the power section would be detrimental to the solenoid operator, or where the operating medium could not be exhausted to the atmosphere.

Size Selection



Q: How do I determine the correct cylinder bore size for my application?

A: Follow these four easy steps:

1. Determine, in pounds, the force needed to do the job. Add 25% for friction and to provide enough power to allow the cylinder rod to move at a reasonable rate of speed.

2. Find out how much air pressure will be used and maintained.

3. Select a power factor from the table below that, when multiplied by the planned air pressure, will produce a force equal to that which was determined in Step 1. The power factor is the mount of square inches for the cylinder bore.

4. The bore diameter that you need will be found directly above the power factor that was determined in Step 3.

Power Factor Table

Bore Diameter:	3/4	1	1 1/8	1 1/2	2	2 1/4	2 1/2	3	3 1/4	4	6
Power Factor:	.4	.8	1.0	1.8	3.1	4.0	4.9	7.1	8.3	12.6	28.3

Example: Estimated force needed is 900 lbs. Air pressure to be used is 80 PSI:

80 PSI x Power Factor = 900 lbs.

Power Factor = 900 lbs / 80 PSI = **11.25**

The power factor just above 11.25 is 12.6. Therefore, this job will require a 4″ bore cylinder.

Piston Rod Strength

If subjected to a heavy load a piston rod may buckle. The following chart suggests minimum rod diameter under various load conditions and when the rod is extended and unsupported and must be used in accordance with the chart's instructions. (see next paragraph). There must be no side load or bend stress at any point along the extending rod.

HOW TO USE THE TABLE: Exposed length of rod is shown at the top of the table. This length is typically longer than the stroke length of the cylinder. The vertical scale shows the load on the cylinder and is in English tons (1 ton = 2000 lbs.) If the rod and and front end of the cylinder barrel are rigidly supported, then a smaller rod will be sufficient; use the column that is 1/2 the length of the actual piston rod. If pivot to pivot mounting is used, double the the actual length of the exposed rod and utilize the suggest rod diameter.

Figures in body of chart are suggested minimum rod diameters

	Exposed Length of Piston Rod (IN)							
Tons	10	20	40	60	70	80	100	120
1/2			3/4	1				
3/4			13/16	1-1/16				
1		5/8	7/8	1-1/8	1-1/4	1-3/8		
1-1/2		11/16	15/16	1-3/16	1-3/8	1-1/2		
2		3/4	1	1-1/4	1-7/16	1-9/16	1-3/16	
3	13/16	7/8	1-1/8	1-3/8	1-9/16	1-5/8	1-7/8	
4	15/16	1	1-3/16	1-1/2	1-5/8	1-3/4	2	2-1/4
5	1	1-1/8	1-5/16	1-9/16	1-3/4	1-7/8	2-1/8	2-3/8
7-1/2	1-3/16	1-1/4	1-7/16	1-3/4	1-7/8	2	2-1/4	2-1/2
10	1-3/8	1-7/16	1-5/8	1-7/8	2	2-1/8	2-7/16	2-3/4
15	1-1/16	1-3/4	1-7/8	2-1/8	2-1/4	2-3/8	2-11/16	3
20	2	2	2-1/8	2-3/8	2-1/2	2-5/8	2-7/8	3-1/4
30	2-3/8	2-7/16	2-1/2	2-3/4	2-3/4	2-7/8	3-1/4	3-1/2
40	2-3/4	2-3/4	2-7/8	3	3	3-1/4	3-1/2	3-3/4
50	3-1/8	3-1/8	3-1/4	3-3/8	3-1/2	3-1/2	3-3/4	4
75	3-3/4	3-3/4	3-7/8	4	4	4-1/8	4-3/8	4-1/2
100	4-3/8	4-3/8	4-3/8	4-1/2	4-3/4	4-3/4	4-7/8	5
150	5-3/8	5-3/8	5-3/8	5-1/2	5-1/2	5-1/2	5-3/4	6

CAUTION: Horizontal or angle mounted cylinders (anything other than vertical) creates a bend stress on the rod when extended, just from the weight of the rod and cylinder itself. Trunnion mounting should be utilized in a position which will balance the cylinder weight when extended.

Pneumatic Cylinder Force

Cylinder forces are shown in pounds for both extension and retraction. Lines standard type show extension forces, using the full piston area. Lines in italic type show retraction forces with various rod sizes. The valves below are theoretical, derived by calculation.

Pressures shown across the top of the chart are differential pressures across the two cylinder ports. In practice, the air supply line must supply another 5% of pressure to make up for cylinder loss, and must supply an estimated 25-50% additional pressure to make up for flow losses in lines and valving so the cylinder will have sufficient travel speed.

For all practical purposes design your system 25% over and above your theoretical calculations.

Piston Dia.	Rod Dia	Effec. Area Sq. In.	60 PSI	70 PSI	80 PSI	90 PSI	100 PSI	110 PSI	120 PSI
1-1/2	None	1.77	106	124	142	159	177	195	230
	5/8	*1.46*	*88*	*102*	*117*	*132*	*146*	*161*	*190*
	1	*0.99*	*59*	*69*	*79*	*89*	*98*	*108*	*128*
2	None	3.14	188	220	251	283	314	345	377
	5/8	*2.83*	*170*	*198*	*227*	*255*	*283*	*312*	*340*
	1	*2.35*	*141*	*165*	*188*	*212*	*235*	*259*	*283*
2-1/2	None	4.91	295	344	393	442	491	540	589
	5/8	*4.60*	*276*	*322*	*368*	*414*	*460*	*506*	*552*
	1	*4.12*	*247*	*289*	*330*	*371*	*412*	*454*	*495*
3	None	7.07	424	495	565	636	707	778	848
	5/8	*6.76*	*406*	*431*	*540*	*608*	*676*	*744*	*814*
3-1/4	None	8.30	498	581	664	747	830	913	996
	1	*7.51*	*451*	*526*	*601*	*676*	*751*	*826*	*902*
	1-3/8	*6.82*	*409*	*477*	*545*	*613*	*681*	*818*	*818*
4	None	12.57	754	880	1006	1131	1257	1283	1508
	1	*11.78*	*707*	*825*	*943*	*1061*	*1178*	*1296*	*1415*
	1-3/8	*11.09*	*665*	*776*	*887*	*998*	*1109*	*1219*	*1330*
5	None	19.64	1178	1375	1571	1768	1964	2160	2357
	1	*18.85*	*1131*	*1320*	*1508*	*1697*	*1885*	*2074*	*2263*
	1-3/8	*18.16*	*1089*	*1271*	*1452*	*1634*	*1816*	*1997*	*2179*
6	None	28.27	1696	1979	2262	2544	2827	3110	3392
	1-3/8	*26.79*	*1607*	*1875*	*2143*	*2411*	*2679*	*2946*	*3214*
	1-3/4	*25.90*	*1552*	*1811*	*2069*	*2328*	*2586*	*2845*	*3104*
8	None	50.27	3016	3519	4022	4524	5027	5530	6032
	1-3/8	*48.79*	*2927*	*3415*	*3903*	*4391*	*4879*	*5366*	*5854*
	1-3/4	*47.90*	*2872*	*3351*	*3829*	*4308*	*4786*	*5265*	*5744*
10	None	78.54	4712	5498	6283	7069	7854	8639	9425
	1-3/4	*76.14*	*4568*	*5329*	*6091*	*6852*	*7614*	*8375*	*9136*
	2	*75.40*	*4524*	*5278*	*6032*	*6786*	*7540*	*8294*	*9048*
12	None	113.10	6786	7917	9048	10179	11310	12441	13572
	2	*110.00*	*6598*	*7697*	*8797*	*9896*	*10996*	*12095*	*13195*
	2-1/2	*108.20*	*6491*	*7573*	*8655*	*9737*	*10819*	*11901*	*12983*

Air Cylinder Speed

Estimating cylinder speed is extremely difficult because of the flow losses within the system in piping, fittings, and porting through the valves which are in the air path. Flow losses cause a loss in pressure which directly effect the force output. To be able to determine the maximum speed of the cylinder, the sum of all flow losses, pressure required for the force output and the available inlet pressure must be known. Circuit losses cannot be determined or calculated accurately. Rules of Thumb are relied upon to determine an approximation of air cylinder speed.

The first general rule of thumb is chose a cylinder which will allow for at least 25% more force then what is required. For extremely fast operations, chose a cylinder which will allow for 50% more force than what is required. This will leave 25% or 50% of inlet pressure to satisfy system losses.

The second rule of thumb is to select a directional control valve which has the same port size as the cylinder which it will be operating. Typically larger valves internal flow capacity is the same as the connection size. On smaller valves ,the internal flow capacity is typically much less than the connection size. Always be sure to check the valves flow rate, and do not relay on the port size.

ESTIMATED CYLINDER SPEED

Figures below are in Inches per Second

	Actual Valve Orifice Dia.							
Bore	1/32	1/16	1/8	1/4	3/8	1/2	3/4	1
1	6	15	37					
1-1/8	5	12	28	85				
1-1/2	3	7	16	50				
2		4	9	28	70			
2-1/2			6	18	45	72		
3			4	12	30	48		
3-1/4			3	10	24	37	79	
4				7	17	28	60	
5				4	11	18	40	82
6				3	7	12	26	55
8					4	7	15	32
10						4	9	20
12						3	6	14

NOTE: These values are an approximate speed, under average conditions, where the force required is 50% of available 80-100 PSI inlet pressure, the directional valve internal flow is equal to the porting and an unlimited supply of air. Acceleration distance is assumed to be relatively short compared to total stroke based upon sufficiently long stroke.

Estimate Travel Speed of Loaded Air Cylinder

Air Flow Through Orifices

The chart below gives theoretical SCFM air flow through sharp edged orifices. In actual practice, approximately 2/3 of this flow is obtained. Assume 75% of line pressure (PSI) is actually working on the load. The remaining 25% is consumed by flow losses in the valve, and connecting lines.

Calculate 75% of your line pressure (PSI) and find it in the first column in the chart below. Move across the table to the column which is the actual port size of your valve. Since valves do not contain sharp edged orifices, divide this number in half.

After finding the SCFM, convert this to CFM at the pressure required to move the load. From this the speed of travel can be estimated.

Approximate SCFM flow though Sharp Edged Orifices

PSI Across Orifice	Orifice Diameter, in Inches										
	1/64	1/32	1/16	1/8	1/4	3/8	1/2	5/8	3/4	7/8	1
5	0.062	0.249	0.993	3.97	15.9	35.7	63.5	99.3	143	195	254
6	0.068	0.272	1.09	4.34	17.4	39.1	69.5	109	156	213	278
7	0.073	0.293	1.17	4.68	18.7	42.2	75.0	117	168	230	300
9	0.083	0.331	1.32	5.30	21.2	47.7	84.7	132	191	260	339
12	0.095	0.379	1.52	6.07	24.3	54.6	97.0	152	218	297	388
15	0.105	0.420	1.68	6.72	26.9	60.5	108	168	242	329	430
20	0.123	0.491	1.96	7.86	31.4	70.7	126	196	283	385	503
25	0.140	0.562	2.25	8.98	35.9	80.9	144	225	323	440	575
30	0.158	0.633	2.53	10.1	40.5	91.1	162	253	365	496	648
35	0.176	0.703	2.81	11.3	45.0	101	180	281	405	551	720
40	0.194	0.774	3.10	12.4	49.6	112	198	310	446	607	793
45	0.211	0.845	3.38	13.5	54.1	122	216	338	487	662	865
50	0.229	0.916	3.66	14.7	58.6	132	235	366	528	718	938
60	0.264	1.06	4.23	16.9	67.6	152	271	423	609	828	1082
70	0.300	1.20	4.79	19.2	76.7	173	307	479	690	939	1227
80	0.335	1.34	5.36	21.4	85.7	193	343	536	771	1050	1371
90	0.370	1.48	5.92	23.7	94.8	213	379	592	853	1161	1516
100	0.406	1.62	6.49	26.0	104	234	415	649	934	1272	1661
110	0.441	1.76	7.05	28.2	113	254	452	705	1016	1383	1806
120	0.476	1.91	7.62	30.5	122	274	488	762	1097	1494	1951
130	0.494	1.98	7.90	31.6	126	284	506	790	1138	1549	2023

Air Consumption Rates

Q: How do I calculate the air consumption of a cylinder?

Example: Determine the air consumption of a 2″ bore cylinder with a 4″ stroke operating 30 complete cycles (out and back) per minute at 80 PSI inlet pressure.

A:

1. Find the area of the piston by converting the bore diameter into square inches.
$(2 \text{ in. bore}/2)^2$ x 3.1416 (∏) = 3.14 sq. in.

2. Determine consumption per single stroke.
3.14 sq. in. x 4 in. stroke = 12.56 cu.in.

3. Determine consumption per complete cycle (Disregard displacement of piston rod because it is generally not significant).
12.56 cu.in. x 2 = 25.12 cu.in. per cycle

4. Determine volume of 80 PSI air that is consumed per minute.
25.12 cu.in. x 30 cycles/minute = 753.6 cu.in./min. of 80 PSI air

5. Convert cu.in. to cu.ft.
$\frac{753.6 \text{ cu.in./min.}}{1728 \text{ cu.in/cu.ft.}} = 0.436$ cu.ft./min.

6. Convert air compressed to 80 PSI to "free" (uncompressed) air.
$\frac{80 \text{ PSI} + 14.7 \text{ PSI}}{14.7 \text{ PSI}} = 6.44$ (times air is compressed when at 80 PSI)

7. Determine cubic feet of free air used per minute.
0.436 cu. ft. x 6.44 compression ratio = 2.81 cu. ft. of free air used per minute

8. So, the consumption rate of a 2″ bore, 4″ stroke cylinder operating 30 complete cycles per minute at 80 PSI is **2.81 SCFM** (Standard Cubic Feet Per Minute) of free air. "Standard" means at a temperature of 70°F and at sea level. Also see questions regarding C_V (pg. 1) and cylinder size selection (pg. 10).

Determine Air Volume Required

The figures in the table below are for cylinders with standard rods. The difference with over-sized rods is negligible. Air consumption was calculated assuming the cylinder would dwell momentarily at the end of each stroke, allowing air to fill up the cylinder to set system pressure. If cylinder strokes prior to allowing for air to fill, air consumption will be less. than what is shown in the table.

Assuming system losses through piping and valves will be approximately 25%, make sure that the cylinder bore selected will balance the load at 75% of the pressure available in the system. Without this surplus pressure the cylinder may not travel at it desired speed.

Using the Table Below

Upon determining the regulator pressure, go to the proper column. The figures below represent a 1″ stroke, extend and retract cycle. Take the figure and multiply times the actual stroke and by the number of cycles needed in one minute. The result will be the SCFM for the application.

Cylinder Air Consumption: 1″ Stroke, Full Cycle

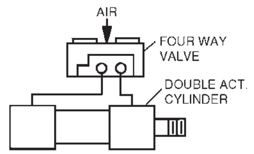
Cylinder Bore	60 PSI	70 PSI	80 PSI	90 PSI	100 PSI	110 PSI	120 PSI	130 PSI	140 PSI	150 PSI
1.50	0.009	0.010	0.012	0.013	0.015	0.016	0.017	0.018	0.020	0.021
2.00	0.018	0.020	0.022	0.025	0.027	0.029	0.032	0.034	0.036	0.039
2.50	0.028	0.032	0.035	0.039	0.043	0.047	0.050	0.054	0.058	0.062
3.00	0.039	0.044	0.050	0.055	0.060	0.066	0.070	0.076	0.081	0.087
3.25	0.046	0.053	0.059	0.065	0.071	0.078	0.084	0.090	0.096	0.102
4.00	0.072	0.081	0.091	0.100	0.110	0.119	0.129	0.139	0.148	0.158
5.00	0.113	0.128	0.143	0.159	0.174	0.189	0.204	0.219	0.234	0.249
6.00	0.162	0.184	0.205	0.227	0.249	0.270	0.292	0.314	0.335	0.357
8.00	0.291	0.330	0.369	0.408	0.447	0.486	0.525	0.564	0.602	0.642
10.00	0.455	0.516	0.576	0.637	0.698	0.759	0.820	0.881	0.940	1.000
12.00	0.656	0.744	0.831	0.919	1.010	1.090	1.180	1.270	1.360	1.450

Example: What is the SCFM of a cylinder in a stamping application, that moves a 2250 lbs. wieght 60 times per minute through a 6″ stoke?

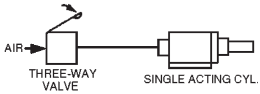
By selecting a 6″ bore, the 2250 lbs. force is realized at 80 PSI. Then add 25% more pressure (20 PSI), to account for system losses and set the regulator at 100 PSI. Then using the table above we have the following calculation:

0.249 x 6 (stroke) x 60 (cycles per minute) = 89.64 SCFM

Double-Acting -vs- Single-Acting



Q: What are the differences between double-acting and single-acting cylinders?
A: Double-acting cylinders provide power on both the "extend" and "retract" stroke. They require the use of four way directional control valves.



Single-acting cylinders provide power only on the "push" stroke. The piston rod is returned by an internal spring. Single-acting cylinders use about one-half as much air as double-acting cylinders and are operated by 3-way valves.

NOTE: Valves Shown here are from the Nova Series (Four-Way Valve) and the MV Series (Three-Way Valve).

Force Output Calculation

Q: How do I figure out the theoretical force output of a cylinder?
A: Follow these steps.

1. Calculate the area of the cylinder piston
   Area = Πr^2
   where Π = 3.1416
   r = 1/2 the bore diameter
2. Multiply the piston area by the air pressure to be used.
   Area x Pressure = Force Output

   Example: What is the theoretical force output of a 2 1/2" bore cylinder operating at 80 lbs. per square inch air pressure?
   Step 1. Area = Πr^2 Area = 3.1416×1.25^2
   Area = 4.91 square inches
   Step 2. 4.91 sq. in. x 80 PSI = 393 lbs. of force

Note: The force output on the rod end of a cylinder will be slightly less due to the displacement of the rod.The real force output of a cylinder will be less than the theoretical output because of internal friction and external side loading.It is best to use a cylinder that will generate from 25% to 50% more force than theoretically needed.

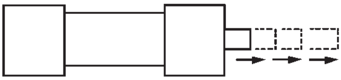
Mid-Stroke Position Sensing

Q: How do I sense the position of a cylinder piston when it is somewhere between its limits?
A: Order your cylinder with Hall Effect or Reed switches and a magnetic piston. Set the switches at the desired trip points. An electrical signal will be emitted when the magnetic piston passes a switch.

Non Lubricated

Q: Are there cylinders available that do not require lubrication?
A: Mead Centaur cylinders have Teflon® seals that glide over the cylinder tube surface without the aid of a lubricant. Other Mead cylinders have a "non-lube" option.

Smoother Cylinder Motion



Q: What could cause a cylinder to move erratically during stroking?

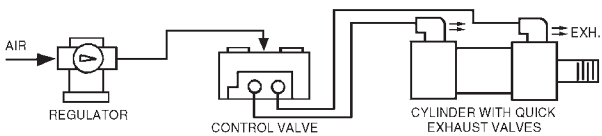
A: Irregular rod motion could be caused by:

1. Too low an input air pressure for the load being moved.
2. Too small a cylinder bore size for the load being moved.
3. Side loading on the cylinder rod caused by misalignment of the rod and load.
4. Using flow control valves to meter the incoming air rather than the exhausting air.
5. Flow control valves are set for too slow a rod movement.
6. An absence of lubrication.

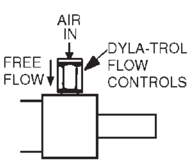
Speed Boost

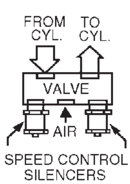
Q: How do I get more speed out of a cylinder?

A: You may increase the inlet pressure to within the recommended limits and/or you may place a quick exhaust valve in either or both cylinder port(s).



Speed Control





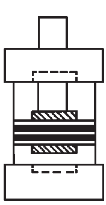
Q: How can I control cylinder speed?

A: Use any of the following methods:

1. Place Mead Dyla-Trol® flow control valves in each cylinder port. Install them so that the air leaving the cylinder is controlled.
2. Use right-angle flow controls in the cylinder ports. These feature recessed screw driver adjustment and convenient swivel for ease of tubing alignment.
3. Place speed control silencers into the exhaust ports of the control valve that is being used to power the cylinder.
4. Purchase a directional valve that has built-in-flow controls. See Mead Dura-Matic Valves.

See Page 7, Flow Controls.

Cushioning



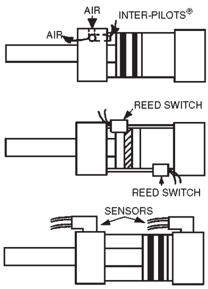
Q: How do I prevent a cylinder from impacting at the end of its stroke?

A: Generally, it is best to order your cylinders with built-in cushions if you anticipate unacceptable end-of-stroke impact. Cushions decelerate the piston rod through the last $\frac{11}{16}$″ of stroke. The degree of cushioning may be adjusted by means of a needle control in the cylinder head.

Mead's DM1, DM2 and HD1 Series cylinders offer adjustable cushion cylinders. Centaur cylinders are all supplied with rubber bumpers at no extra charge. Adjustable cushions and bumpers eliminate the "clank" that occurs at stroke completion.

Position Sensing, End-Of-Stroke

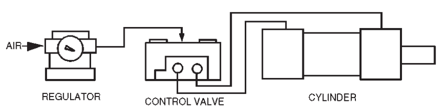
Q: How do I sense that a cylinder rod has reached the end of its stroke?
A: Use any of the following methods or external limit valves:



1. Order your cylinder with Inter-Pilots®. A built-in, normally closed, 3-way valve that emits an air signal when the stroke limit is reached. Inter-Pilots® are available on the DM1, DM2 and HD1 cylinders. Note: To use Inter-Pilots®, the full stroke of the cylinder must be used.
2. Order your cylinder with Hall Effect or Reed switches that emit electrical signals when the stroke limit is reached. Note: To use Hall Effect or Reed switches, the cylinder must be supplied with a magnetic piston.
3. Use stroke completion sensors. These valves react to pressure drops so that an output signal will be generated even if the piston is stopped short of a complete stroke.

Increasing Power

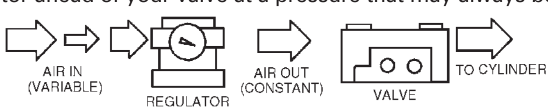
Q: How do I get more power out of a particular cylinder?
A: You should increase the pressure of the air that feeds the cylinder within the recommended limits.



NOTE: The Control Valve shown here is from the Nova Series.

Pressure Maintenance

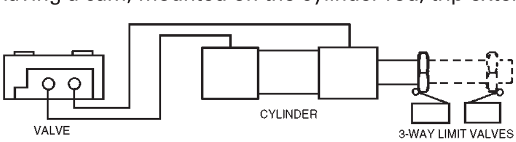
Q: How do I maintain a constant cylinder force output when my air pressure supply fluctuates?
A: Set an air regulator ahead of your valve at a pressure that may always be maintained.



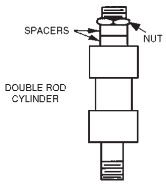
Example: Depending on the time of day and workload, a plant's air pressure fluctuates between 80 and 95 PSI. Set the regulator at 80 PSI and the cylinder power output through the plant will remain constant. Also, an air reservoir may be used to solve an air shortage problem. By mounting a reservoir close to a cylinder, an adequate amount of air will be supplied when needed.

Reciprocating

Q: How do I get a cylinder to reciprocate automatically?
A: Order your cylinder with Inter-Pilots®, Hall Effect or Reed switches, or stroke completion sensors . These devices will send signals to double pressure or solenoid operated valves that will shift each time a stroke has been completed. Reciprocation may also be achieved by having a cam, mounted on the cylinder rod, trip external limit valves.



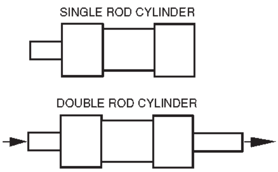
NOTE: The Valve shown here is from the Nova Series. The 3-Way Limit Valves are from the MV series.



Adjustable Stroke

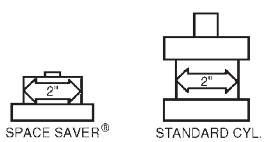
Q: Is it possible to make the stroke of a cylinder adjustable?
A: Yes. Double-acting cylinders may be ordered with a common rod that protrudes from both cylinder end caps. A nut may be placed on one rod end to retain spacers that will limit the stroke distance. Be sure to guard the spacer end because “pinch points” will be present. DM1, DM2 and HD1 Cylinders are double acting.

Single and Double Rods



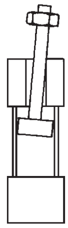
Q: What is the difference between a single and double-rod cylinder?
A: Single-rod cylinders have a piston rod protruding from only one end of the cylinder. Double-rod cylinders have a common rod, driven by a single piston, protruding from both cylinder end caps. When one end retracts, the other extends. They are excellent for providing an adjustable stroke and providing additional rigidity. Also, a double-rod with attached cam may be used to trip a limit switch.

Space Conserving Type



Q: I have a space problem and cannot fit a regular cylinder into the area available. What can I do?
A: Use the ultra-compact Mead “Space-Saver” cylinder.

Side Load Reduction



Q: How may I minimize the adverse effects of cylinder side loading?
A: **First**, be sure that the object being moved is in exact alignment with the piston rod. if the cylinder is rigidly mounted and the rod is forced off line, the cylinder bearing will wear prematurely and a loss of power will occur. It may be helpful to use guide rails to keep the object being moved in proper alignment.
Second, don’t use all of the stroke. Particularly on pivot and clevis models, it is wise to have the piston stop a few inches short of full stroke. This makes the cylinder more rigid and extends bearing life.
Third, order your cylinder with an external bearing. An external bearing takes advantage of physics by providing more bearing surface and a longer lever point than a standard cylinder type. (Order HD1 (Heavy Duty Air Cylinders)) **Fourth**, Order a Self Aligning Rod Coupler.

The Table on the right shows the Rod Couplers that Mead offers. The thread shown is a male / female thread as the coupler has both a male and female end.

Rod Couplers

Part #	Rod Thread
DMA-312	5/16-24
DMA-375	3/8-24
DMA-437	7/16-20
DMA-500	1/2-20
DMA-625	5/8-18
DMA-750	3/4-16
DMA-875	7/8-14
DMA-1000	1-14
DMA-1250	1 1/4-12

High Temperature Operations

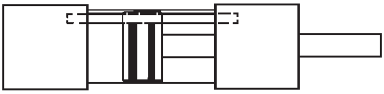
Q: I have an application in a high temperature environment. What should I do to avoid complication?
A: The control valve powering the cylinder should be mounted as far away from the heat as possible. While temperatures exceeding 100°C (212°F) can cause breakdown in Buna N seals, most of Mead's cylinders may be supplied with fluorocarbon seals instead of Buna N. Fluorocarbon seals are effective to 204°C (400°F). Flurocarbon seals are also known as Viton® seals.

Non-Lubricated Air Circuit

Q: Is it possible to build an air circuit using components that do not require lubrication?
A: Mead Micro-Line pilot valves (MV), Capsula directional valves, and Centaur cylinders will provide excellent service without lubrication. Most Mead cylinders are available with optional non-lube seals.

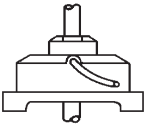
Cylinder Presses, Non-Rotating

Q: How do I prevent the tooling attached to my air press rod from turning?
A: Order the press cylinder with a non-rotating rod option.



CYLINDER WITH NON-ROTATING ROD

Collet Fixtures



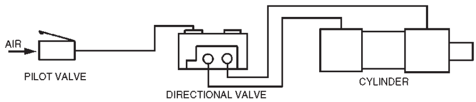
Q: Is there a way of firmly holding smooth round bars with an air powered device?
A: Use an air collet fixture. The device operates just like a double acting cylinder; air to close and open. The collet fixture uses standard industrial collets and can not only handle round bars but also hex bars.

Mead offers a 5C and 3C Collet fixture (Models LS-1 and PCF).

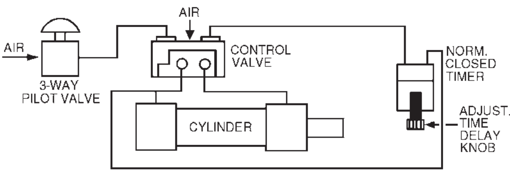
Basic Control Circuits

Air Circuitry

Q: What is a typical air circuit?
A: The simplest and most common air circuit consists of a double-acting cylinder which is controlled by a four-way directional valve. The directional valve is actuated by air pilot valves or electric switches.

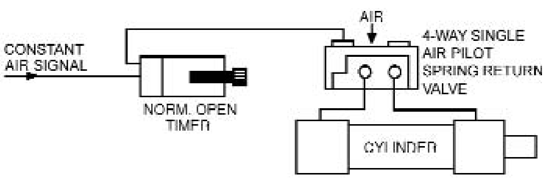


Timing Circuits



Sample Components
3-Way Air Pilot - MV-140
Control VAlve - N2-DP
Normally Closed Timer - KLC-105

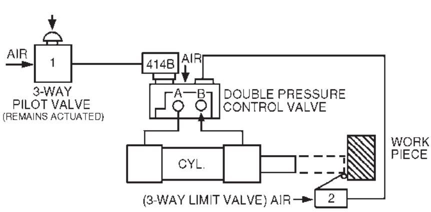
In this circuit, the 3-way valve is actuated and air is sent to the control valve. The control valve shifts, sending air to the rear of the cylinder causing the cylinder to extend. Air also flows to the timer where it begins to time to the pre-setting. Once reached, the timer opens, allowing the air to flow through to the control valves other pilot port, shifting the valve back. Air flows through port B, retracting the cylinder.



Sample Components
Normally Open Timer - KLH-105
Control Valve - N2-SP

In this circuit a constant air signal is sent to the timer. The normally open timer allows air to flow through until the set time period expires. While air flows to the pilot of the control valve the cylinder extends and remains extended. When the time period expires the cylinder returns even if the air signal remains. NOTE: In this set-up if the air signal is removed before the timer, the cylinder will retract. The circuit will only recycle once the air signal is removed and re-applied.

Dual Signal Circuit



Sample Components
3-Way Pilot Valve - MV-140
Control Valve - N2-DP
Impulse Relay - 414B
3-Way Limit Valve - MV-15

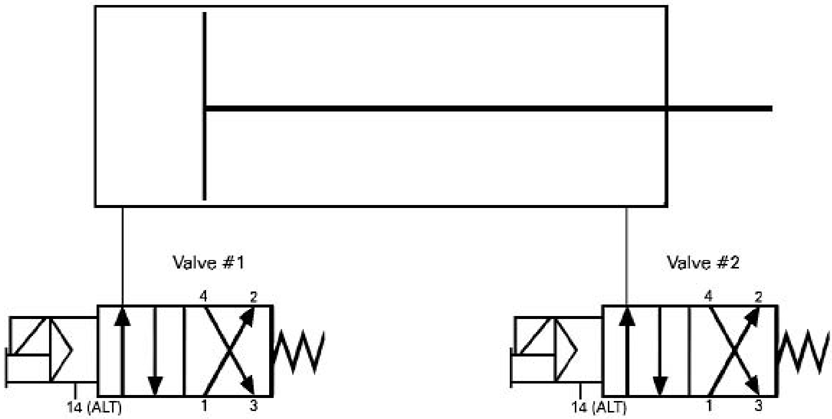
When actuated, the 3-way valve sends a signal to 414B, which emits a signal to the control valve. The 3-way valve remains actuated. The valve shifts, allowing air to flow through port A, extending the cylinder. 414B senses the back pressure caused by the shifted valve, closes, and exhausts. Since the signal from valve #1 is blocked by the closed 414B, valve #2 (when actuated) shifts the control valve back. Air flows through port B, retracting the cylinder.

2 Valves for 3 Position Function

Use these set-ups to obtain a Three Position Function with (2) Two Position valves. The circuitry shown is ideal for use with the Isonic product line.

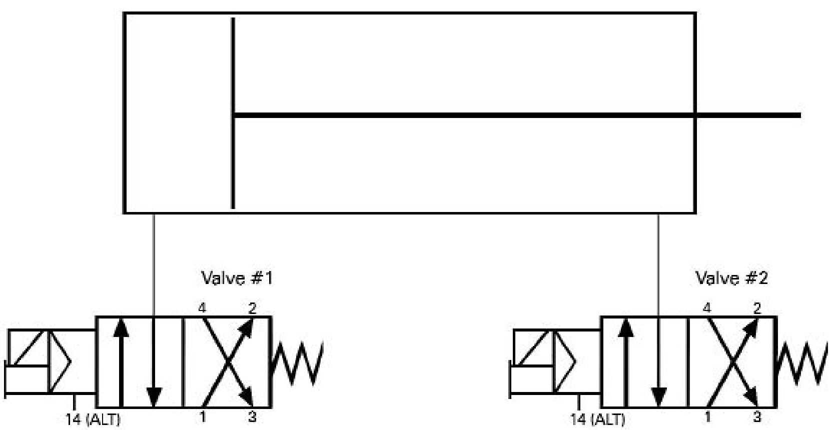
Pressure Applied Set-Up

Actuate Valve #1 for Retraction; Actuate Vavle #2 for Extension.
Supply pressure must be equal on both valve #1 and #2.



Pressure Relieved Set-Up

Actuate Valve #1 for Extension; Actuate Valve #2 for Retraction.
Supply pressure does not have to be equal.

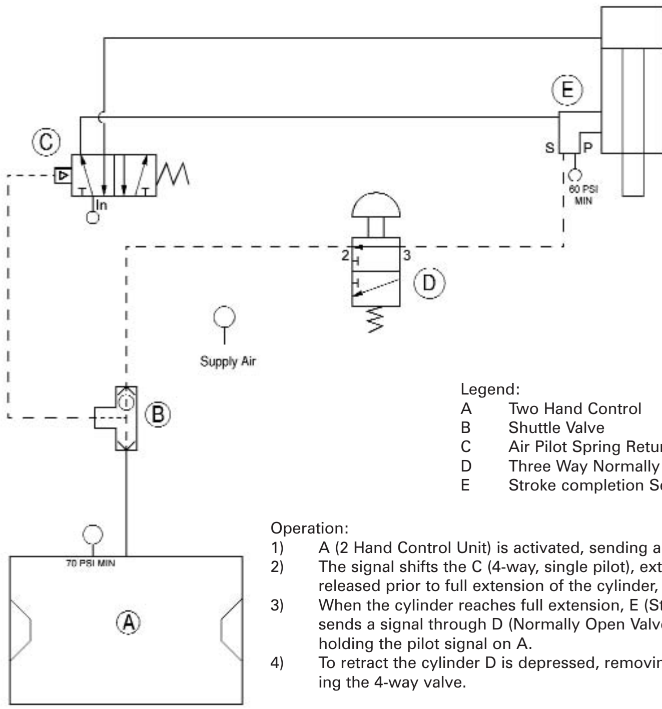


For an All Ports Blocked Three Position Function, an additional 2-way valve must be used as for blocking the exhaust of the two valves. This 3rd valve is actuated when ever either one of the other valves is actuated. Contact the Mead to discuss further application set-ups.

Two Hand Extend One Hand Retract

For applications where a secondary operation must occur, utilize this circuit. This circuit allows for the operator to be “tied down” during the clamping of a part via the actuation of the two hand control. Once the rod movement has stopped, the operator can the move onto the secondary operation.

Additionally with the use of the stroke completion sensor the circuit will work even if clamping on material that is not consistently the same size.



Legend:
A Two Hand Control
B Shuttle Valve
C Air Pilot Spring Return 4 Way Control Valve
D Three Way Normally Open Push Button
E Stroke completion Sensor

Operation:
1) A (2 Hand Control Unit) is activated, sending a signal through shuttle valve.
2) The signal shifts the C (4-way, single pilot), extending the cylinder. If A is released prior to full extension of the cylinder, the cylinder will retract.
3) When the cylinder reaches full extension, E (Stroke Completion Sensor) sends a signal through D (Normally Open Valve), through B (Shuttle valve), holding the pilot signal on A.
4) To retract the cylinder D is depressed, removing the pilot signal from C, shift ing the 4-way valve.

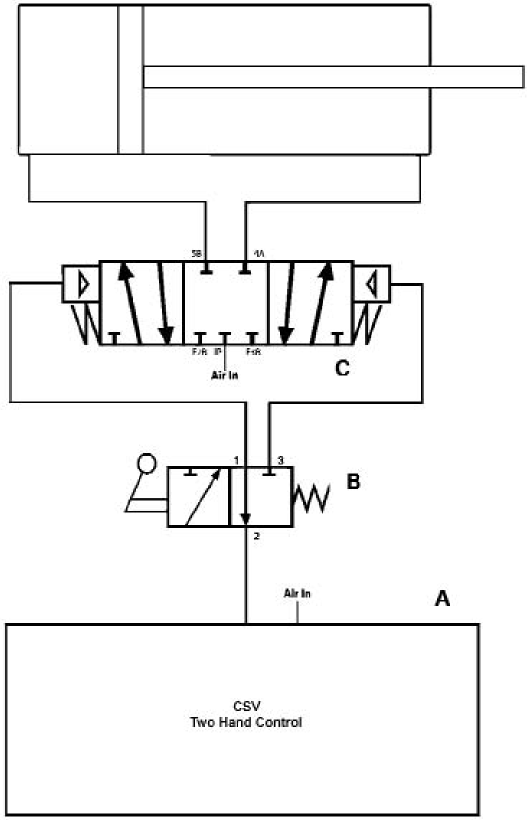
The Bill Of Materials to the left can be used to mix and match for your specific application. Additionally multiple components maybe added at “D”. (Example: Timer and Push Button combination for an automatic return or manual return.)

A	CSV-101	2 Hand Anti- Tie Down Control Unit
	CSV-101 LS	Same as above, but with low stress buttons
	CSV 107 LS1	Same as CSV-101, but w/ remote buttons
	CSV 107 LS2	Same as above but/ with low force actuators
B	SV-1	Shuttle valve
C	N2-SP	¼” port spring return
	C2-3	¼” port spring return, rugged applications
	C5-3	½” port spring return, rugged applications
D	MV-140	Spring return three-way valve
	MV-ES	Emergency Stop
	KLH-105	Timer 1-10 sec.
	MV-	Any MV- type Valve will work here, set up Normally Open
E	SCS-112	1/8” Stroke Completion Sensor (SCS)
	SCS-250	¼” SCS
	SCS-375	3/8” SCS
	SCS-500	½” SCS

Two Hand Extend Two Hand Retract

Use this circuit, where a “pinch point” exists on both the extension and retraction of the linear actuator. This circuit will require the operator to use the two hand control for either motion.

The suggested components will accommodate up to one 4” bore cylinder with relatively good speed. If a larger bore cylinder is used or more air volume is required, contact Mead.



Operation:

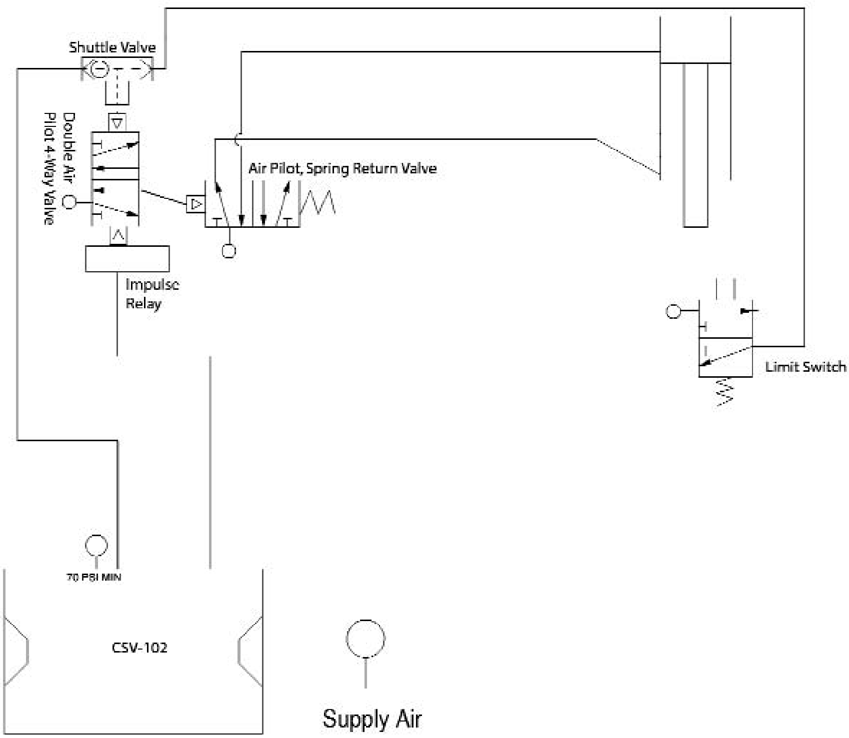
1) Operator sets “B” valve to either extend or retract cylinder.
2) Operator uses “A” (two hand control) to move cylinder.
3) If one or both buttons are not actuated cylinder will stop in place.

A	CSV-101 CSV-101 LS CSV 107 LS1 CSV 107 LS2	2 Hand Anti- Tie Down Control Unit Same as above, but with low stress buttons Same as CSV-101, but w/ remote buttons Same as above but/ with low force actuators
B	MV-35 MV-TP	Two Position Detented 3-Way Valve Two Position Detented 3-Way Valve
C	C2-2H	Three Position Spring Centered 4-Way Valve

2 Hand Extend with Automatic Return

This Circuit is useful for applications where cycle time and safety is an issue. With the Automatic Return feature, the operators hands are tied down and the cylinder will return when the work is completed, not when the operator removes their hands from the actuator.

Operator uses CSV-102 (Two Hand Control) to the extend cylinder, if one or both hands are removed, cylinder returns. If limit is reached the cylinder will auto return even if the operators hand remain on the two hand control.



CSV-102 when actuated, pilots the Double Air Pilot 4-Way Valve to allow air to the Air Pilot Spring Return Valve. When released the CSV-102, pilots the Double Air Pilot 4-Way Valve back to the original position. The Impulse Relay takes the constant input from the CSV-102 and changes it to an impulse allowing for the auto-return from the Limit Switch.

Bill Of Material With Typical Mead Components

Component	Mead Part Number
CSV-102	CSV-102
Impulse Relay	414B
Double Air Pilot 4-Way Valve	N2-DP
Shuttle Valve	SV-1
Air Pilot, Spring Return Valve	N2-SP
Limit Switch	MV Type

The suggested components will accommodate up to a 4" Bore Cylinder. Contact Mead if your application requires a larger bore cylinder.

Pneumatic Pipe Size

The pipe sizes listed in the chart below are assuming a 100 PSI pneumatic system to carry air at a 1 PSI loss per 100 feet. Conservatively figure each pipe fitting to equal 5 feet of pipe. At pressures other than 100 PSI, flow capacity will be inversely proportionate to pressure (Calculated by Boyle's Law and based upon absolute PSIA Pressure levels).

SCFM Flow	Length of Run - Feet 25	50	75	100	150	200	300	500	100	Compressor HP
6	1/2	1/2	1/2	1/2	1/2	1/2	1/2	3/4	3/4	1
18	1/2	1/2	1/2	3/4	3/4	3/4	3/4	1	1	3
30	3/4	3/4	3/4	3/4	1	1	1	1-1/4	1-1/4	5
45	3/4	3/4	1	1	1	1	1-1/4	1-1/4	1-1/4	7-1/2
60	3/4	1	1	1	1-1/4	1-1/4	1-1/4	1-1/2	1-1/2	10
90	1	1	1-1/4	1-1/4	1-1/4	1-1/4	1-1/2	1-1/2	2	15
120	1	1-1/4	1-1/4	1-1/4	1-1/2	1-1/2	1-1/2	2	2	20
150	1-1/4	1-1/4	1-1/4	1-1/2	1-1/2	2	2	2	2-1/2	25
180	1-1/4	1-1/2	1-1/2	1-1/2	2	2	2	2-1/2	2-1/2	30
240	1-1/4	1-1/2	1-1/2	2	2	2	2-1/2	2-1/2	3	40
300	1-1/2	2	2	2	2	2-1/2	2-1/2	3	3	50
360	1-1/2	2	2	2	2-1/2	2-1/2	2-1/2	3	3	60
450	2	2	2	2-1/2	2-1/2	3	3	3	3-1/2	75
600	2	2-1/2	2-1/2	2-1/2	3	3	3	3-1/2	4	100
750	2	2-1/2	2-1/2	3	3	3	3-1/2	3-1/2	4	125

Pneumatic Pressure Loss

Figures in the table below are approximate PSI compressed air pressure losses for every 100 feet of clean commercial steel pipe. (Schedule 40)

CFM Free Air	1/2 INCH 80 PSI	1/2 INCH 125 PSI	3/4 INCH 80 PSI	3/4 INCH 125 PSI	1 INCH 80 PSI	1 INCH 125 PSI	1-1/4 INCH 80 PSI	1-1/4 INCH 125 PSI	1-1/2 INCH 80 PSI	1-1/2 INCH 125 PSI
10	0.45	0.30	0.11	0.08	0.04	0.02				
20	1.75	1.15	0.40	0.28	0.15	0.08				
30	3.85	2.55	0.90	0.60	0.30	0.20				
40	6.95	4.55	1.55	1.05	0.45	0.30				
50	10.50	7.00	2.40	1.60	0.75	0.50	0.18	0.12		
60			3.45	2.35	1.00	0.70	0.25	0.17		
70			4.75	3.15	1.35	0.90	0.35	0.23	0.16	0.10
80			6.15	4.10	1.75	1.20	0.45	0.30	0.20	0.14
90			7.75	5.15	2.25	1.50	0.56	0.40	0.25	0.17
100			9.60	6.35	2.70	1.80	0.65	0.45	0.30	0.20
125			15.50	9.80	4.20	2.80	1.05	0.70	0.45	0.32
150			23.00	14.50	5.75	4.00	1.45	1.00	0.65	0.45
175					8.10	5.45	2.00	1.30	0.90	0.60
200					10.90	7.10	2.60	1.75	1.15	0.80
250							4.05	2.65	1.80	1.20
300							5.80	3.85	2.55	1.70
350							7.90	5.15	3.55	2.35
400							10.30	6.75	4.55	3.05
450									5.80	3.80
500									7.10	4.70

Air Flow Loss Through Pipes

Instructions: Find the factor from the chart below according to the pipe size and SCFM. Divide the factor by the Compression Ratio. Then multiply the number by the actual length of pipe, in feet, then divide by 1000. This result is the pressure loss in PSI.

Compression Ratio = (Gauge Pressure + 14.5) / 14.5

Pressure Loss (PSI) = Factor / Compression Ratio x Length of Pipe (Ft) / 1000

Factor Table

	Pipe Size NPT							
SCFM	1/2	3/4	1	1-1/4	1-1/2	1-3/4	2	2-1/2
5	12.7	1.2	0.5					
10	50.7	7.8	2.2	0.5				
15	114	17.6	4.9	1.1				
20	202	30.4	8.7	2.0				
25	316	50.0	13.6	3.2	1.4	0.7		
30	456	70.4	19.6	4.5	2.0	1.1		
35	621	95.9	26.6	6.2	2.7	1.4		
40	811	125	34.8	8.1	3.6	1.9		
45		159	44.0	10.2	4.5	2.4	1.2	
50		196	54.4	12.6	5.6	2.9	1.5	
60		282	78.3	18.2	8.0	4.2	2.2	
70		385	106	24.7	10.9	5.7	2.9	1.1
80		503	139	32.3	14.3	7.5	3.8	1.5
90		646	176	40.9	18.1	9.5	4.8	1.9
100		785	217	50.5	22.3	11.7	6.0	2.3
110		950	263	61.1	27.0	14.1	7.2	2.8
120			318	72.7	32.2	16.8	8.6	3.3
130			369	85.3	37.8	19.7	10.1	3.9
140			426	98.9	43.8	22.9	11.7	1.4
150			490	113	50.3	26.3	13.4	5.2
160			570	129	57.2	29.9	15.3	5.9
170			628	146	64.6	33.7	17.6	6.7
180			705	163	72.6	37.9	19.4	7.5
190			785	177	80.7	42.2	21.5	8.4
200			870	202	89.4	46.7	23.9	9.3
220				244	108	56.5	28.9	11.3
240				291	128	67.3	34.4	13.4
260				341	151	79.0	40.3	15.7
280				395	175	91.6	46.8	18.2
300				454	201	105	53.7	20.9

Pressure Loss Through Pipe Fittings

This chart gives figures that are the air pressure flow losses through screw fittings expressed in the equivalent lengths of straight pipe of the same diameter. For example, a 2" gate valve flow resistance would be the same as 1.3 foot of straight pipe.

Pipe Size NPT	Gate Valve	Long Radius Elbow*	Medium Radius Elbow**	Standard Elbow***	Angel Valve	Close Return Bend	Tee Thru Side	Globe Valve
1/2	0.31	0.41	0.52	0.84	1.1	1.3	1.7	2.5
3/4	0.44	0.57	0.73	1.2	1.6	1.8	2.3	3.5
1	0.57	0.77	0.98	1.6	2.1	2.3	3.1	4.7
1-1/4	0.82	1.1	1.4	2.2	2.9	3.3	4.4	6.5
1-1/2	0.98	1.3	1.6	2.6	3.5	3.9	5.2	7.8
2	1.3	1.7	2.2	3.6	4.8	5.3	7.1	10.6
2-1/2	1.6	2.2	2.8	4.4	5.9	6.6	8.7	13.1
3	2.1	3.0	3.6	5.7	7.7	8.5	11.4	17.1
4	3.0	3.9	5.0	7.9	10.7	11.8	15.8	23.7
5	3.9	5.1	6.5	10.4	13.9	15.5	20.7	31

* or run of Standard Tee
** or run of tee reduced in size by 25%
*** or run of tee reduced in size by 50%

Friction of Air in Hose

Pressure Drop per 25 feet of hose. Factors are proportionate for longer or shorter lengths.

Size	SCFM	50 PSI	60 PSI	70 PSI	80 PSI	90 PSI	100 PSI	110 PSI
1/2" ID	20	1.8	1.3	1	0.9	0.8	0.7	0.6
	30	5	4	3.4	2.8	2.4	2.3	2
	40	10.1	8.4	7	6	5.4	4.8	4.3
	50	18.1	14.8	12.4	10.8	9.5	8.4	7.6
	60		23.4	20	17.4	14.8	13.3	12
	70			28.4	25.2	22	19.3	17.6
	80				34.6	30.5	27.2	24.6
3/4" ID	20	0.4	0.3	0.2	0.2	0.2	0.2	0.1
	30	0.8	0.6	0.5	0.5	0.4	0.4	0.3
	40	1.5	1.2	0.9	0.8	0.7	0.6	0.5
	50	2.4	1.9	1.5	1.3	1.1	1	0.9
	60	3.5	2.8	2.3	1.9	1.6	1.4	1.3
	70	4.4	3.8	3.2	2.8	2.3	2	1.8
	80	6.5	5.2	4.2	3.6	3.1	2.7	2.4
	90	8.5	6.8	5.5	4.7	4	3.5	3.1
	100	11.4	8.6	7	5.8	5	4.4	3.9
	110	14.2	11.2	8.8	7.2	6.2	5.4	4.9
1" ID	30	0.2	0.2	0.1	0.1	0.1	0.1	0.1
	40	0.3	0.3	0.2	0.2	0.2	0.2	0.2
	50	0.5	0.4	0.4	0.3	0.3	0.2	0.2
	60	0.8	0.6	0.5	0.5	0.4	0.4	0.3
	70	1.1	0.8	0.7	0.7	0.6	0.5	0.4
	80	1.5	1.2	1	0.8	0.7	0.6	0.5
	90	2	1	1.3	1.1	0.9	0.8	0.7
	100	2.6	2	1.6	1.4	1.2	1	0.9
	110	3.5	2.6	2	1.7	1.4	1.2	1.1

Vacuum Flow Trough Orifices

The chart below approximates flow that would be expected thorugh a practical orifice. Flows are 2/3 of theoretical value obtained through a sharp edged orifice.

NOTE: Multiple smaller holes size grippers will work more efficently at higher vacuums.

Chart Valves are Air Flows in SCFM

Orifice Dia., Inches	Degree of Vacuum Across Orifice, Inches of Mercury (Hg)								
	2"	4"	6"	8"	10"	12"	14"	18"	24"
1/64	0.018	0.026	0.032	0.037	0.041	0.045	0.048	0.055	0.063
1/32	0.074	0.100	0.128	0.148	0.165	0.180	0.195	0.220	0.250
1/16	0.300	0.420	0.517	0.595	0.660	0.725	0.780	0.880	1.00
1/8	1.2	1.68	2.06	2.37	2.64	2.89	3.12	3.53	4.04
1/4	4.8	6.7	8.3	9.5	10.6	11.6	12.4	14.0	16.2
3/8	10.8	15.2	18.5	21.4	23.8	26.0	28.0	31.8	36.4
1/2	19.1	27.0	33.0	38.5	42.3	46.3	50.0	56.5	64.6
5/8	30.0	42.2	51.7	59.5	66.2	72.6	78.0	88.0	101
3/4	43.0	60.6	74.0	85.3	95.2	104	112	127	145
7/8	58.8	82.6	101	116	130	142	153	173	198
1	76.5	108	131	152	169	185	200	225	258

Decimal Equivalents

(of Fraction, Wire Gauge and Metric Sizes)

Sizes	Decimal Inches	Sizes	Decimal Inches	Sizes	Decimal Inches	Sizes	Decimal Inches
107	0.0019	.7mm	0.0276	1.95mm	0.0768	3.7mm	0.1457
106	0.0023	70	0.0280	5/64	0.0781	26	0.1470
105	0.0027	69	0.0292	47	0.0785	3.75mm	0.1476
104	0.0031	.75mm	0.0295	2mm	0.0787	25	0.1495
103	0.0035	68	0.0310	2.05mm	0.0807	3.8mm	0.1496
102	0.0039	1/32	0.0312	46	0.0810	24	0.1520
101	0.0043	.8mm	0.0315	45	0.0820	3.9mm	0.1535
100	0.0047	67	0.0320	2.1mm	0.0827	23	0.1540
99	0.0051	66	0.0330	2.15mm	0.0846	5/32	0.1562
98	0.0055	.85mm	0.0335	44	0.0860	22	0.1570
97	0.0059	65	0.0350	2.2mm	0.0866	4mm	0.1575
96	0.0063	.9mm	0.0354	2.25mm	0.0886	21	0.1590
95	0.0067	64	0.0360	43	0.0890	20	0.1610
94	0.0071	63	0.0370	2.3mm	0.0906	4.1mm	0.1614
93	0.0075	.95mm	0.0374	2.35mm	0.0925	4.2mm	0.1654
92	0.0079	62	0.0380	42	0.0935	19	0.1660
.2mm	0.0079	61	0.0390	3/32	0.0938	4.25mm	0.1673
91	0.0083	1mm	0.0394	2.4mm	0.0945	4.3mm	0.1693
90	0.0087	60	0.0400	41	0.0960	18	0.1695
.22mm	0.0087	59	0.0410	2.45mm	0.0965	11/64	0.1719
89	0.0091	1.05	0.0413	40	0.0980	17	0.1730
88	0.0095	58	0.0420	2.5mm	0.0984	4.4mm	0.1732
.25mm	0.0098	57	0.0430	39	0.0995	16	0.1770
87	0.0100	1.1mm	0.0433	38	0.1015	4.5mm	0.1772
86	0.0105	1.15mm	0.0453	2.6mm	0.1024	15	0.1800
85	0.0110	56	0.0465	37	0.1040	4.6mm	0.1811
.28mm	0.0110	3/64	0.0469	2.7mm	0.1063	14	0.1820
84	0.0115	1.2mm	0.0472	36	0.1065	13	0.1850
.3mm	0.0118	1.25mm	0.0492	2.75mm	0.1083	4.7mm	0.1850
83	0.0120	1.3mm	0.0512	7/64	0.1094	4.75mm	0.1870
82	0.0125	55	0.0520	35	0.1100	3/16	0.1875
.32mm	0.0126	1.35mm	0.0531	2.8mm	0.1102	4.8mm	0.1890
81	0.0130	54	0.0550	34	0.1110	12	0.1890
80	0.0135	1.4mm	0.0551	33	0.1130	11	0.1910
.35mm	0.0138	1.45mm	0.0571	2.9mm	0.1142	4.9mm	0.1929
79	0.0145	1.5mm	0.0591	32	0.1160	10	0.1935
1/64	0.0156	53	0.0595	3mm	0.1181	9	0.1960
.4mm	0.0157	1.55mm	0.0610	31	0.1200	5mm	0.1969
78	0.0160	1/16	0.0625	3.1mm	0.1220	8	0.1990
.45mm	0.0177	1.6mm	0.0630	1/8	0.1250	5.1mm	0.2008
77	0.0180	52	0.0635	3.2mm	0.1260	7	0.2010
.5mm	0.0197	1.65mm	0.0650	3.25mm	0.1280	13/64	0.2031
76	0.0200	1.7mm	0.0669	30	0.1285	6	0.2040
75	0.0210	51	0.0670	3.3mm	0.1299	5.2mm	0.2047
.55mm	0.0217	1.75mm	0.0689	3.4mm	0.1339	5	0.2055
74	0.0225	50	0.0700	29	0.1360	5.25mm	0.2067
.6mm	0.0236	1.8mm	0.0709	3.5mm	0.1378	5.3mm	0.2087
73	0.0240	1.85mm	0.0728	28	0.1405	4	0.2090
72	0.0250	49	0.0730	9/64	0.1406	5.4mm	0.2126
.65mm	0.0256	1.9mm	0.0748	3.6mm	0.1417	3	0.2130
71	0.0260	48	0.0760	27	0.1440	5.5mm	0.2165

Decimal Equivalents

(of Fraction, Wire Gauge and Metric Sizes)

Sizes	Decimal Inches	Sizes	Decimal Inches	Sizes	Decimal Inches	Sizes	Decimal Inches
7/32	0.2188	7.4mm	0.2913	V	0.3770	21/32	0.6562
5.6mm	0.2205	M	0.2950	9.6mm	0.3780	17mm	0.6693
2	0.2211	7.5mm	0.2953	9.7mm	0.3819	43/64	0.6719
5.7mm	0.2244	19/64	0.2969	9.75mm	0.3839	11/16	0.6875
5.75mm	0.2264	7.6mm	0.2992	9.8mm	0.3858	17.5mm	0.6890
1	0.2280	N	0.3020	W	0.3860	45/64	0.7031
5.8mm	0.2283	7.7mm	0.3031	9.9mm	0.3898	18mm	0.7087
5.9mm	0.2323	7.75mm	0.3051	25/64	0.3906	23/32	0.7188
A	0.2340	7.8mm	0.3071	10mm	0.3937	18.5mm	0.7283
15/64	0.2344	7.9mm	0.3110	X	0.3970	47/64	0.7344
6mm	0.2362	5/16	0.3125	Y	0.4040	19mm	0.7480
B	0.2380	8mm	0.3150	13/32	0.4062	3/4	0.7500
6.1mm	0.2402	O	0.3160	Z	0.4130	49/64	0.7656
C	0.2420	8.1mm	0.3189	10.5mm	0.4134	19.5mm	0.7677
6.2mm	0.2441	8.2mm	0.3228	27/64	0.4219	25/32	0.7812
D	0.2460	P	0.3230	11mm	0.4331	20mm	0.7874
6.25mm	0.2461	8.25mm	0.3248	7/16	0.4375	51/64	0.7969
6.3mm	0.2480	8.3mm	0.3268	11.5mm	0.4528	20.5mm	0.8071
E	0.2500	21/64	0.3281	29/64	0.4531	13/16	0.8125
1/4	0.2500	8.4mm	0.3307	15/32	0.4688	21mm	0.8268
6.4mm	0.2520	Q	0.3320	12mm	0.4724	53/64	0.8281
6.5mm	0.2559	8.5mm	0.3346	31/64	0.4844	27/32	0.8438
F	0.2570	8.6mm	0.3386	12.5mm	0.4921	21.5mm	0.8465
6.6mm	0.2598	R	0.3390	1/2	0.5000	55/64	0.8594
G	0.2610	8.7mm	0.3425	13mm	0.5118	22mm	0.8661
6.7mm	0.2638	11/32	0.3438	33/64	0.5156	7/8	0.8750
17/64	0.2656	8.75mm	0.3445	17/32	0.5312	22.5mm	0.8858
6.75mm	0.2657	8.8mm	0.3465	13.5mm	0.5315	57/64	0.8906
H	0.2660	S	0.3480	35/64	0.5469	23mm	0.9055
6.8mm	0.2677	8.9mm	0.3504	14mm	0.5512	29/32	0.9062
6.9mm	0.2717	9mm	0.3543	9/16	0.5625	59/64	0.9219
I	0.2720	T	0.3580	14.5mm	0.5709	23.5mm	0.9252
7mm	0.2756	9.1mm	0.3583	37/64	0.5781	15/16	0.9375
J	0.2770	23/64	0.3594	15mm	0.5906	24mm	0.9449
7.1mm	0.2795	9.2mm	0.3622	19/32	0.5938	61/64	0.9531
K	0.2810	9.25mm	0.3642	39/64	0.6094	24.5mm	0.9646
9/32	0.2812	9.3mm	0.3661	15.5mm	0.6102	31/32	0.9688
7.2mm	0.2835	U	0.3680	5/8	0.6250	25mm	0.9843
7.25mm	0.2854	9.4mm	0.3701	16mm	0.6299	63/64	0.9844
7.3mm	0.2874	9.5mm	0.3740	41/64	0.6406	1	1.0000
L	0.2900	3/8	0.3750	16.5mm	0.6496		

Conversions Between US Units (English) and SI Units (Metric)

Quantity	US Unit	SI Unit	Coversion Factor
Length	inch (in.)	millimeter (mm)	1 in. = 25.4mm
Pressure*	pounds / sq. in.	bar	1 bar = 14.5 PSI
Vacuum**	inches of mercury (in. Hg)	mm of mercury (mm Hg)	1" Hg = 25.4mm Hg
Flow***	cubic feet per minute (cfm)	cubic decimeters per sec (dm3/sec)	2.12 cfm = 1 dm3/sec
Force	pound (f) or lb. (f)	Newton (N)	1 lb (f) = 4.44 N
Mass	pound (m) or lb. (m)	kilogram (Kg)	1 Kg = 2.2 lbs
Volume****	gallon (US gallon)	liter (l)	1 US Gal = 3.71 l
Temperature	degrees Fahrenheit (°F)	degrees Celsius (°C)	°C = 5/9 (°F-32)
Torque	pounds (f) - inches (lbs (f) - in.)	Newton-meters (Nm)	1 Nm = 8.88 lb(f)-in.
Power	horsepower (HP)	kilowatt (kw)	1 kw = 1.34 HP
Frequency	cycles per second (cps)	Hertz (Hz)	1 Hz = 1 cps
Velocity	feet per second (fps)	meter per second (m/s)	1 m/s = 3.28 fps

*Above Atmospheric (PSI or Bar); **Below Atmospheric (Hg); ***Gas; (f) = force; (m) = mass

Interchange Tables

How to Use: The following charts interchange units from the SI International Standard, the US system (or English System) and older metric systems. The left column is the basic SI unit. Equivalents are in the same line. To best use these charts, find the unit that is to be converted and move to the row with the "1" in it. Move in the same row to the unit you are changing the value to and multiply by that number to make the conversion.

Torque

Newton-Meters	Kilopond-Meters	Foot-lbs	Inch-lbs
1	1.020×10^{-1}	7.376×10^{-1}	8.851
0.01	1	7.233	86.80
1.356	1.382×10^{-1}	1	12
1.130×10^{-1}	1.52×10^{-2}	8.333×10^{-2}	1

Gravity Due to Acceleration

US System (g) = 32.2 feet per sec. per sec.
Metric System (g) = 105.5 meters per sec. per sec

Length

(Linear Measurement)

Meter	Centimeter	Kilometer	Mile	Inch	Foot
1	100	$1x10^{4}$	$6.214x10^{-4}$	39.370	3.281
0.01	1	$1x10^{5}$	$6.214x10^{-6}$	$3.937x10^{-1}$	$3.281x10^{-2}$
$1x10^{-3}$	0.10	$1x10^{-6}$	$6.214x10^{-7}$	$3.937x10^{-2}$	$3.281x10^{-3}$
$1x10^{3}$	$1x10^{5}$	1	$6.214x10^{-7}$	$3.937x10^{4}$	$3.281x10^{3}$
$1.609x10^{3}$	$1.609x10^{5}$	1.609	1	$6.336x10^{4}$	5280
$2.540x10^{-2}$	2.540	$2.540x10^{-5}$	$1.578x10^{-5}$	1	$8.333x10^{-2}$
$3.048x10^{-1}$	30.479	$3.048x10^{-4}$	$1.894x10^{-4}$	12	1

1 mm = 0.001 m = 0.10 cm = 0.000001 km = 0.03937 in = 0.003281 ft

AREA

(Square Measurement)

Square Meter	Sq. Centimeter	Sq. Kilometer	Square Inch	Square Foot	Square Mile
1	$1x10^{4}$	$1x10^{6}$	$1.550x10^{3}$	10.764	$3.861x10^{-7}$
$1x10^{-3}$	1	$1x10^{-10}$	$1.550x10^{-1}$	$1.076x10^{-3}$	$3.861x10^{-11}$
$1x10^{-6}$	$1x10^{-2}$	$1x10^{-12}$	$1.550x10^{3}$	$1.076x10^{-5}$	$3.861x10^{-13}$
$1x10^{6}$	$1x10^{10}$	1	$1.550x10^{9}$	$1.076x10^{7}$	$3.861x10^{-1}$
$6.452x10^{-4}$	6.452	$6.452x10^{-10}$	1	$6.944x10^{-3}$	$2.491x10^{-10}$
$9.290x10^{-2}$	$9.290x10^{2}$	$9.290x10^{-8}$	144	1	$3.587x10^{-8}$
$2.590x10^{6}$	$2.590x10^{10}$	2.590	$2.788x10^{7}$	$2.788x10^{7}$	1

1 sq. mm = 0.000001 sq. m = 0.00155 sq. in. = 0.00001076 sq. ft

Volume (Cubic)

Cubic Meter	Cu. Decimeter	Cu. Centimeter	US Gallon	Cu. Inch	Cubic Foot
1	$1x10^{3}$	$1x10^{6}$	$2.642x10^{2}$	$6.102x10^{4}$	35.314
$1x10^{-3}$	1	$1x10^{3}$	$2.642x10^{-1}$	61.024	$3.531x10^{-2}$
$1x10^{-6}$	$1x10^{-3}$	1	$2.642x10^{-4}$	$6.102x10^{2}$	$3.531x10^{-5}$
$4.546x10^{-3}$	4.546	$4.546x10^{3}$	1.200	$2.774x10^{2}$	$1.605x10^{-1}$
$3.785x10^{-3}$	3.785	$3.785x10^{3}$	1	$2.310x10^{2}$	$1.337x10^{-1}$
$1.639x10^{-5}$	$1.639x10^{-2}$	16.387	$4.329x10^{-3}$	1	$5.787x10^{-4}$
$2.832x10^{-2}$	28.317	$2.832x10^{4}$	7.481	$1.728x10^{3}$	1

1 imperial gallon = 1.2 US gallon = 0.004546 cu. meter = 4.546 liter = 4546 cu. centimeters

Force (Including Force due to Weight)

Newton	Dyne	Kilopond	Metric Ton	US Ton	Pound
1	$1x10^{5}$	$1.020x10^{-1}$	$1.020x10^{-4}$	$1.124x10^{-4}$	$2.248x10^{-1}$
$1x10^{5}$	1	$1.020x10^{-6}$	$1.020x10^{-9}$	$1.124x10^{-9}$	$2.248x10^{-6}$
9.807	$9.807x10^{5}$	1	$1x10^{-3}$	$1.102x10^{-3}$	2.205
$9.807x10^{3}$	$9.807x10^{8}$	1000	1	1.102	$2.205x10^{3}$
$9.964x10^{3}$	$9.964x10^{8}$	$1.016x10^{2}$	1.016	1.120	$2.240x10^{3}$
$8.896x10^{3}$	$8.896x10^{8}$	$9.072x10^{2}$	$9.072x10^{-1}$	1	2000
4.448	$4.448x10^{5}$	$4.536x10^{-1}$	$4.536x10^{-4}$	$5x10^{-4}$	1

1 long ton = 9964 Newtons = 1016 Kiloponds = 1.016 metric tons = 1.120 US tons = 2240 pounds

Mass (Not Weight)

Kilogram	Gram	Metric Ton	Newton	Pound	US Ton
1	1000	$1x10^{-3}$	9.807	2.205	$1.102x10^{-3}$
$1x10^{-3}$	1	$1x10^{-6}$	$9.807x10^{-3}$	$2.205x10^{-3}$	$1.102x10^{6}$
$1x10^{3}$	$1x10^{6}$	1	$9.807x10^{3}$	$2.205x10^{3}$	1.102
$1.020x10^{-1}$	$1.020x10^{2}$	$1.020x10^{-4}$	1	$2.248x10^{-1}$	$1.120x10^{-4}$
$4.536x10^{-1}$	$4.536x10^{2}$	$4.536x10^{-4}$	4.448	1	$5x10^{-4}$
14.594	$1.459x10^{4}$	$1.459x10^{-2}$	$1.431x10^{-2}$	32.170	$1.609x10^{-2}$
$9.072x10^{2}$	$9.072x10^{5}$	$9.072x10^{-1}$	$8.896x10^{3}$	2000	1

Unit Pressure (Either Fluid or Mechanical)

Bar	Newton/m^{2} (Pascal)	Kilopond/m^{2}	Atmosphere	Pounds/Ft^{2}	Pounds/$Inch^{2}$ (PSI)
$1x10^{-5}$	1	$1.020x10^{-1}$	$9.869x10^{6}$	$2.088x10^{-2}$	$1.45x10^{-4}$
1	$1x10^{6}$	$1.020x10^{4}$	$9.869x10^{-1}$	$2.088x10^{3}$	14.5
$9.807x10^{-5}$	9.807	1	$9.678x10^{-5}$	$2.048x10^{-1}$	$1.422x10^{-3}$
$9.807x10^{-1}$	$9.807x10^{4}$	$1x10^{4}$	$9.678x10^{-1}$	$2.048x10^{3}$	14.220
1.013	$1.013x10^{5}$	$1.033x10^{4}$	1	$2.116x10^{3}$	14.693
$4.789x10^{-4}$	47.893	4.884	$4.726x10^{-4}$	1	$6.944x10^{-3}$
$6.897x10^{-2}$	$6.897x10^{3}$	$7.033x10^{2}$	$6.806x10^{-2}$	$1.440x10^{2}$	1

1 kiloponds / sq cm = 0.9807 bar = 98070 Pascal = 0.9678 atmos = 2048 lbs / sq ft = 14.22 lbs / sq in

Velocity

Meters / Second	Kilometers / Hour	Miles / Hour	Feet / Minute	Feet / Second	Inches / Minute
1	3.6	2.237	$1.968x10^{2}$	3.281	$2.362x10^{3}$
$1x10^{-1}$	$1x10^{-4}$	$6.214x10^{-5}$	$5.468x10^{-3}$	$9.113x10^{-5}$	$6.562x10^{-2}$
$2.778x10^{-1}$	1	$6.214x10^{-1}$	$5.468x10^{-1}$	$9.113x10^{-1}$	$6.562x10^{2}$
$4.470x10^{-1}$	1.609	1	88	1.467	$1.056x10^{3}$
$5.080x10^{-3}$	$1.829x10^{-2}$	$1.136x10^{-2}$	1	$1.667x10^{-2}$	12
$3.048x10^{-1}$	1.097	$6.818x10^{-1}$	60	1	$7.2x10^{2}$
$4.233x10^{-4}$	$1.524x10^{-3}$	$9.470x10^{-4}$	$8.333x10^{-2}$	$1.389x10^{-3}$	1

1 decimeter / second = 0.1 meters / second = 0.005468 feet / minute = 0.06562 inches / minute

Notes

Notes

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